

Model Reduction and Mode Computation for Damped Resonant MEMS

David Bindel

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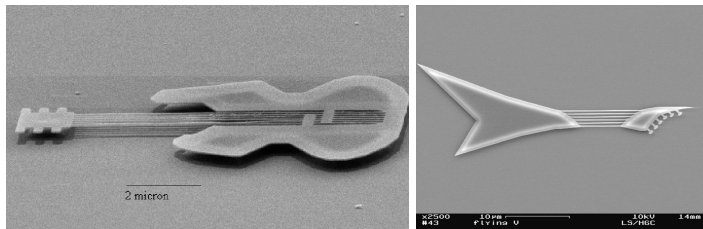
Collaborators

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- Tsuyoshi Koyama
- Sanjay Govindjee
- Sunil Bhave
- Emmanuel Quévy
- Zhaojun Bai

Resonant MEMS

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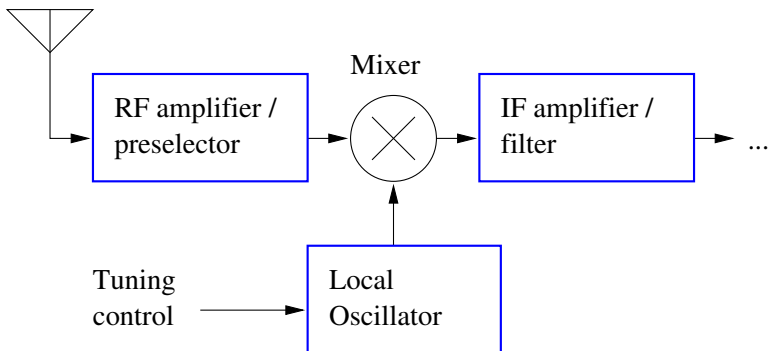


Microguitars from Cornell University (1997 and 2003)

- MHz-GHz mechanical resonators
- Favorite application: radio on chip
- Close second: really high-pitch guitars

The Mechanical Cell Phone

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- Your cell phone has many moving parts!
- What if we replace them with integrated MEMS?

Ultimate Success

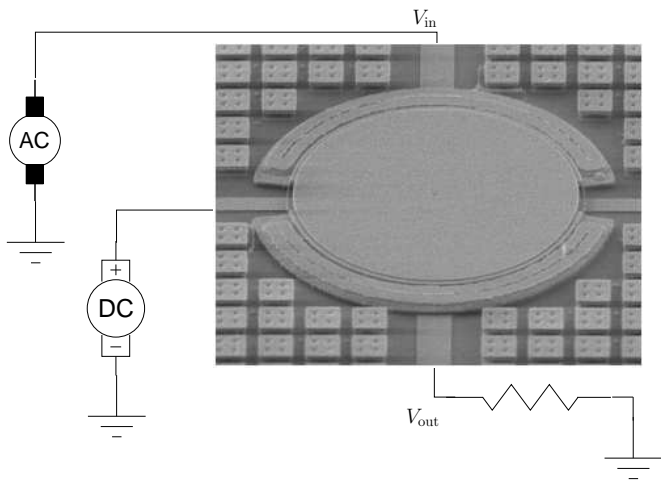
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“Calling Dick Tracy!”



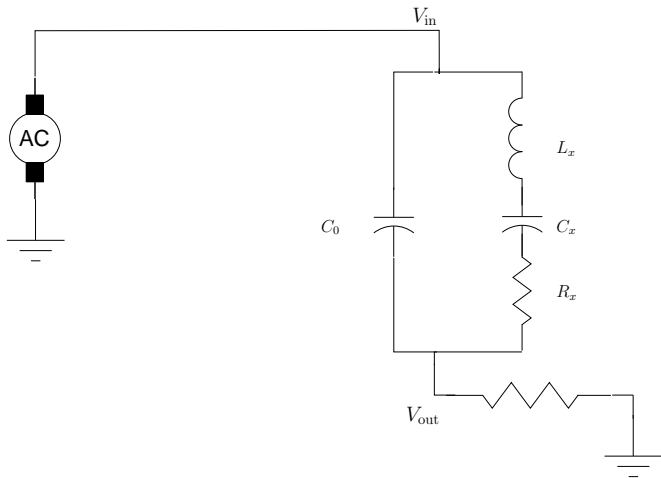
Example Resonant System

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Example Resonant System

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The Designer's Dream

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Ideally, would like

- Simple models for behavioral simulation
- Parameterized for design optimization
- Including all relevant physics
- With reasonably fast and accurate set-up

We aren't there yet. Today, some progress on the last two.

Damping and Q

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Designers want high *quality of resonance* (Q)

- Dimensionless damping in a one-dof system

$$\frac{d^2 u}{dt^2} + Q^{-1} \frac{du}{dt} + u = F(t)$$

- For a resonant mode with frequency $\omega \in \mathbb{C}$:

$$Q := \frac{|\omega|}{2 \operatorname{Im}(\omega)} = \frac{\text{Stored energy}}{\text{Energy loss per radian}}$$

Damping Mechanisms

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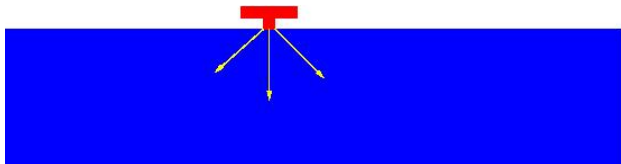
Possible loss mechanisms:

- Anchor loss
- Thermoelastic damping
- Other material losses
- Fluid damping

Our goal: Reduced models that include these effects.

Perfectly Matched Layers

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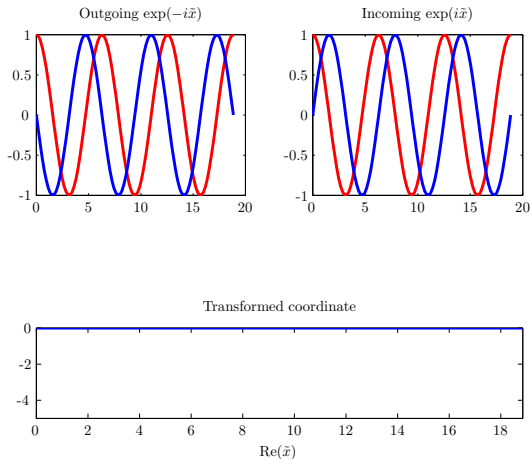
Model substrate as semi-infinite with a

Perfectly Matched Layer (PML).

- Complex coordinate transformation
- Generates a “perfectly matched” absorbing layer
- Idea works with general linear wave equations

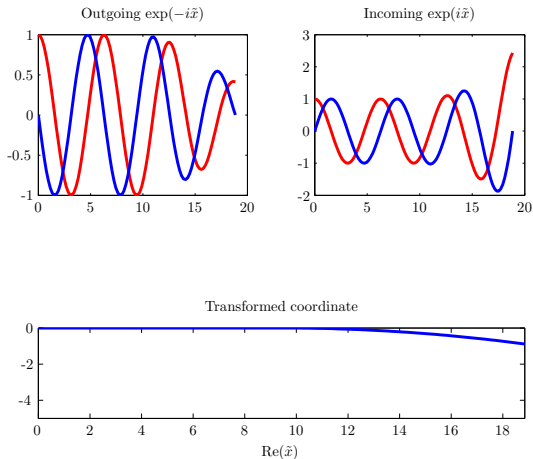
Perfectly Matched Layers by Picture

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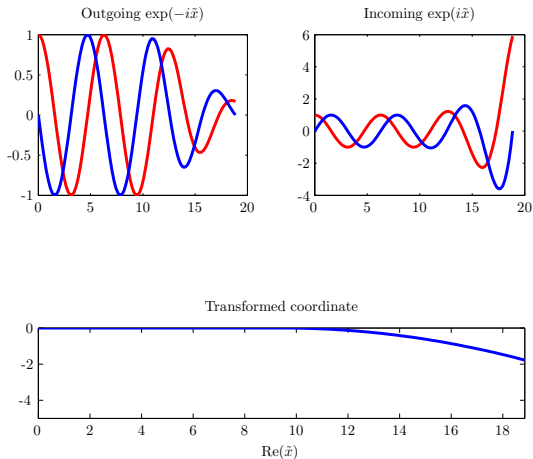
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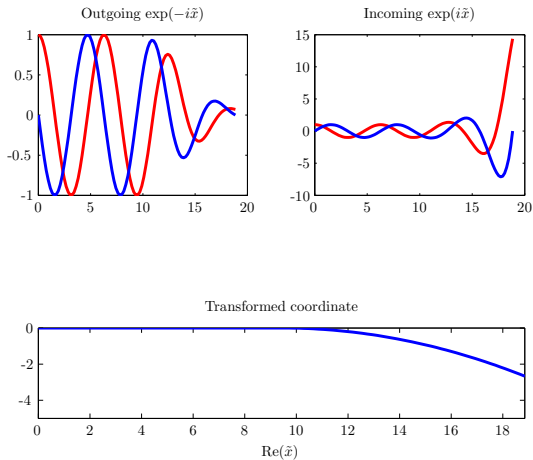
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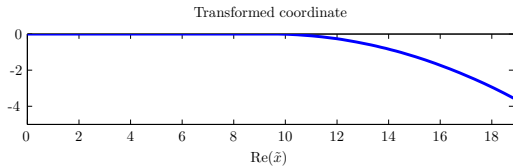
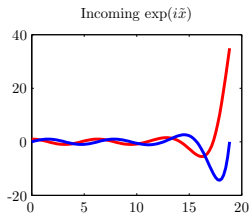
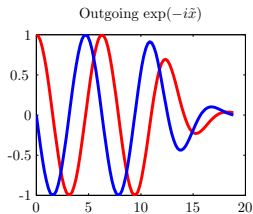
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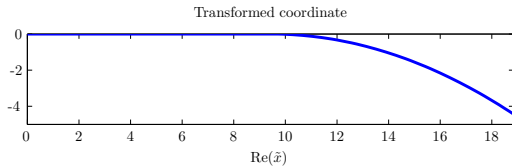
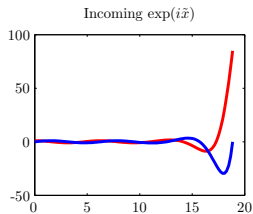
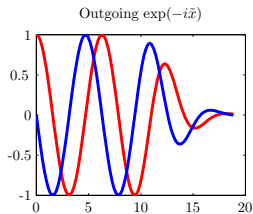
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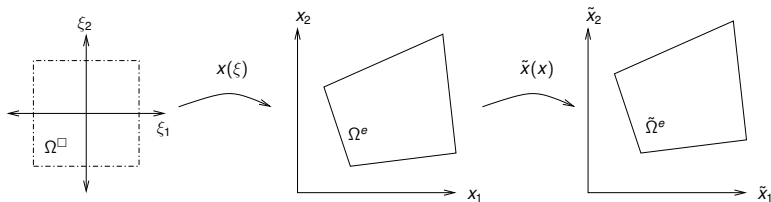
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Finite Element Implementation

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- Combine PML and isoparametric mappings

$$\mathbf{k}^e = \int_{\Omega^\square} \tilde{\mathbf{B}}^T \mathbf{D} \tilde{\mathbf{B}} \tilde{J} d\Omega^\square$$

$$\mathbf{m}^e = \int_{\Omega^\square} \rho \mathbf{N}^T \mathbf{N} \tilde{J} d\Omega^\square$$

- Matrices are *complex symmetric*

Complex Symmetry

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Discretized (forced) problem + fixed PML take the form:

$$(K - \omega^2 M)u = f, \text{ where } K = K^T, M = M^T$$

Can still characterize u as a stationary point of

$$I(u) = \frac{1}{2}u^T(K - \omega^2 M)u - u^T f.$$

Eigenvalues of (K, M) are stationary points of

$$\rho(u) = \frac{u^T K u}{u^T M u}$$

First-order accurate vectors \implies
second-order accurate eigenvalues.

Accurate Model Reduction

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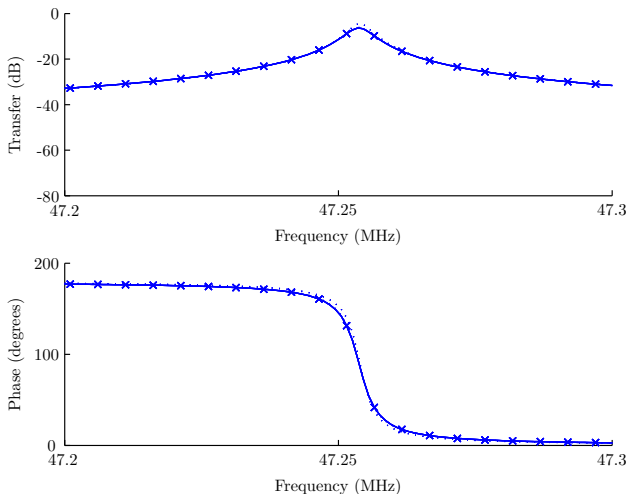
- Usual: Orthogonal projection onto Arnoldi basis V .
- Us: Build new projection basis from V :

$$W = \text{orth}[\text{Re}(V), \text{Im}(V)]$$

- $\text{span}(W)$ contains both \mathcal{K}_n and $\bar{\mathcal{K}}_n$
 \implies double digits correct vs. projection with V
- W is a real-valued basis
 \implies projected system is complex symmetric

Model Reduction Accuracy

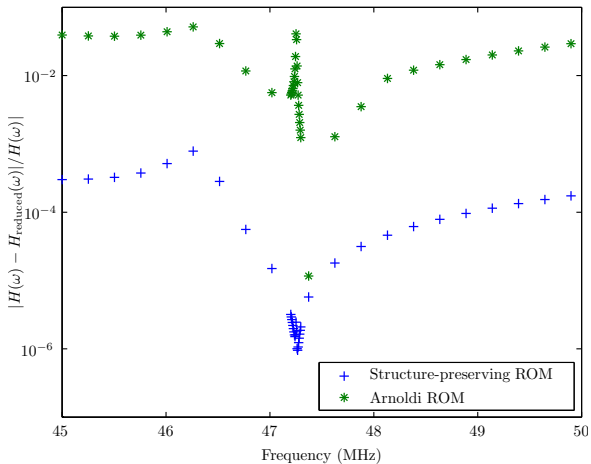
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Results from ROM (solid and dotted lines) near indistinguishable from full model (crosses)

Model Reduction Accuracy

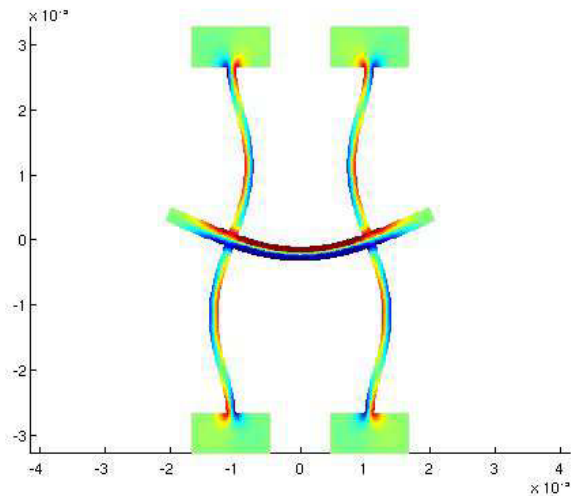
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Preserve structure \implies
get twice the correct digits

Thermoelastic Damping (TED)

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Thermoelastic Damping (TED)

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u is displacement and $T = T_0 + \theta$ is temperature

$$\begin{aligned}\sigma &= \mathbf{C}\epsilon - \beta\theta\mathbf{1} \\ \rho\ddot{u} &= \nabla \cdot \sigma \\ \rho c_v \dot{\theta} &= \nabla \cdot (\kappa \nabla \theta) - \beta T_0 \text{tr}(\dot{\epsilon})\end{aligned}$$

- Coupling between temperature and volumetric strain:
 - Compression and expansion \implies heating and cooling
 - Heat diffusion \implies mechanical damping
 - Not often an important factor at the macro scale
 - Recognized source of damping in microresonators
- Zener: semi-analytical approximation for TED in beams
- We consider the fully coupled system

Nondimensionalized Equations

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Continuum equations:

$$\begin{aligned}\sigma &= \hat{\mathbf{C}}\epsilon - \xi\theta\mathbf{1} \\ \ddot{\mathbf{u}} &= \nabla \cdot \sigma \\ \dot{\theta} &= \eta\nabla^2\theta - \text{tr}(\dot{\epsilon})\end{aligned}$$

Discrete equations:

$$\begin{aligned}M_{uu}\ddot{\mathbf{u}} + K_{uu}\mathbf{u} &= \xi K_{u\theta}\theta + \mathbf{f} \\ C_{\theta\theta}\ddot{\theta} + \eta K_{\theta\theta}\theta &= -C_{\theta u}\dot{\mathbf{u}}\end{aligned}$$

- Micron-scale poly-Si devices: ξ and η are $\sim 10^{-4}$.
- Linearize about $\xi = 0$

Perturbative Mode Calculation

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Discretized mode equation:

$$\begin{aligned}(-\omega^2 M_{uu} + K_{uu})u &= \xi K_{u\theta}\theta \\ (i\omega C_{\theta\theta} + \eta K_{\theta\theta})\theta &= -i\omega C_{\theta u}u\end{aligned}$$

First approximation about $\xi = 0$:

$$\begin{aligned}(-\omega_0^2 M_{uu} + K_{uu})u_0 &= 0 \\ (i\omega_0 C_{\theta\theta} + \eta K_{\theta\theta})\theta_0 &= -i\omega_0 C_{\theta u}u_0\end{aligned}$$

First-order correction in ξ :

$$-\delta(\omega^2)M_{uu}u_0 + (-\omega_0^2 M_{uu} + K_{uu})\delta u = \xi K_{u\theta}\theta_0$$

Multiply by u_0^T :

$$\delta(\omega^2) = -\xi \left(\frac{u_0^T K_{u\theta}\theta_0}{u_0^T M_{uu}u_0} \right)$$

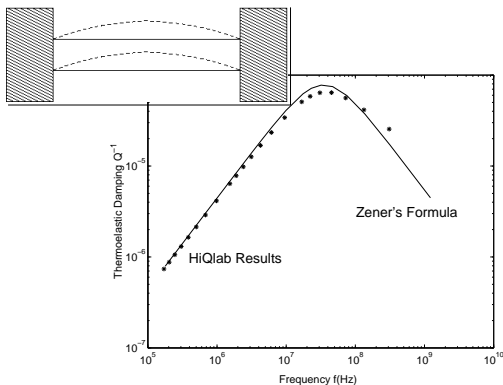
Zener's Model

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- 1 Clarence Zener investigated TED in late 30s-early 40s.
- 2 Model for beams common in MEMS literature.
- 3 “Method of orthogonal thermodynamic potentials” == perturbation method + a variational method.

Comparison to Zener's Model

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- Comparison of fully coupled simulation to Zener approximation over a range of frequencies
- Real and imaginary parts after first-order correction agree to about three digits with Arnoldi

General Picture

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If $w^* A = 0$ and $Av = 0$ then

$$\delta(w^* Av) = w^*(\delta A)v$$

This implies

- If $A = A(\lambda)$ and $w = w(\nu)$, have

$$w^*(\nu)A(\rho(\nu))\nu = 0.$$

ρ stationary when $(\rho(\nu), \nu)$ is a nonlinear eigenpair.

- If $A(\lambda, \xi)$ and w_0^* and v_0 are null vectors for $A(\lambda_0, \xi_0)$,

$$w_0^*(A_\lambda \delta\lambda + A_\xi \delta\xi)v_0 = 0.$$

Conclusions

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- Resonant MEMS have lots of interesting applications
- Designers want reduced models with relevant physics
- Damping is crucial, but not well handled in general
- Our work: use equation structure in making reduced models with damping (modal or more general)